AMENDMENTS TO THE SPECIFICATION:

Page 1, please add the following <u>new paragraphs</u> before paragraph [0001]:

[0000.2] CROSS-REFERENCE TO RELATED APPLICATIONS

[0000.4] This application is a 35 USC 371 application of PCT/DE 2004/001994 filed on September 6, 2004.

[0000.6] BACKGROUND OF THE INVENTION

[0000.8] Field of the Invention

Please replace paragraph [0001] with the following amended paragraph:

[0001] The invention relates to a valve for a fuel injection system of an internal combustion engine[[,]] having the characteristics recited in the preamble to claim 1, specifically and in particular for an injector of a common rail injection system.

Please replace paragraph [0002] with the following amended paragraph:

[0002] Description of the Prior Art

Please replace paragraph [0003] with the following amended paragraph:

[0003] Common rail injection systems have a plurality of injectors, which are supplied with fuel from a central high-pressure reservoir, known as a common rail, by a high-pressure pump under the control of an electronic engine controller, and which inject the fuel via a valve into the combustion chambers of the cylinders of the internal combustion engine. Once such valve is known, from among other sources German Patent Disclosure DE 199 40 296 A1 of the present Applicant and, depending on the valve position, serves to connect a high-pressure region of an injector of the injection system with a low-pressure region, or to disconnect

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them, when fuel is injected through the valve into the combustion chamber of a cylinder and

when the delivery of fuel is to be interrupted, respectively.

Page 2, please replace paragraph [0006] with the following amended paragraph:

[0006] Advantages of the Invention

SUMMARY AND ADVANTAGES OF THE INVENTION

Please replace paragraph [0007] with the following amended paragraph:

[0007] By comparison, with the use of the valve of the invention, having the characteristics

recited in claim 1, cavitation damage can be prevented with good success, since the fuel

stream downstream of the valve seat is not deflected only simply in the axial direction.

Instead, on passing through [[the]] a hollow throat, it is imparted a speed component in a

direction that points away from the center axis of the valve member, so that after emerging

from the hollow throat, it strikes a diametrically opposed region of an inner wall of an

outflow bore of the valve housing. On impact, some of the fuel stream is directed along the

inner wall back in the direction of the valve gap, and as a result, immediately downstream of

this gap, an eddy forms in the widened annular chamber between the hollow throat and the

diametrically opposite wall region of the inner wall. As a result of this eddy, on the one hand

additional fuel is introduced into the annular chamber downstream of the valve gap, so that

more fuel is present there, which counteracts cavitation phenomena in the vicinity of the

valve gap and as a result counteracts cavitation damage at the valve seat that is caused over

the long term. On the other hand, the fuel directed back in the direction of the valve gap

flows along the inner wall of the valve housing, so that additional fuel is introduced precisely

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into this region that is especially threatened with cavitation, and local vapor bubble formation

as a consequence of a fuel pressure drop can be avoided.

Page 3, please replace paragraph [0010] with the following amended paragraph:

[0010] While the outer circumferential surface portion, adjoining the edge on the side toward

the cross-sectional thickening, is preferably oriented essentially parallel to a center axis of the

valve member, the circumferential surface portion adjoining the edge on the side toward the

hollow throat is preferably inclined counter to the flow direction at an angle of between 20°

and 80°, preferably between 30° and 60°, to the center axis of the valve member, so that the

two circumferential surface portions meet one another at an angle of between 200° and 260°,

and preferably between [[190]] 200° and 240°.

Page 5, please replace paragraph [0014] with the following amended paragraph:

[0014] Drawing BRIEF DESCRIPTION OF THE DRAWINGS

Please replace paragraph [0015] with the following amended paragraph:

[0015] The invention will be described in further detail below in terms of an exemplary

embodiment, in conjunction with the associated drawings, in which [[.]] Shown are:

Please replace paragraph [0016] with the following amended paragraph:

[0016] Fig. 1[[, a]] is a fragmentary side view of a valve member or valve bolt of a valve of

the invention;

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Please replace paragraph [0018] with the following amended paragraph:

[0018] Fig. 3, an enlarged detail of Fig. 2, but with a different geometry of the valve member

downstream of the valve gap in terms of the flow direction; and

Please replace paragraph [0020] with the following amended paragraph:

[0020] Description of the Exemplary Embodiments

DESCRIPTION OF THE PREFERRED EMBODIMENTS

Page 6, please replace paragraph [0022] with the following amended paragraph:

[0022] The complete structure of such an injector is described at length, for example in

German Patent Disclosure DE 196 19 523 A1 of the present Applicant, while further details

of the structure of its valve can be found in the aforementioned DE 199 40 296 A1 of the

present Applicant; further explanation is therefore dispensed with at this point, and for such

explanation, see these references.

Please replace paragraph [0023] with the following amended paragraph:

[0023] The valve 2 substantially comprises a valve housing 4, into which a rotationally

symmetrical valve bolt 6 (see Fig. 1) is inserted axially movably. The valve bolt 6 has a

conical sealing face 8, which tapers in the flow direction and which when the valve 2 is

closed rests sealingly against a complementary conical valve seat 10 of the housing 4. As

best shown in Figs. 2 through 4, when the valve 2 is open the sealing face 8 together with the

valve seat 10 defines define a valve gap 12, surrounding the valve bolt 6, in the form of an

annular flow conduit, through which the fuel to be injected flows from the high-pressure side

14 of the valve 2 to its low-pressure side 16.

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Page 9, please replace paragraph [0028] with the following amended paragraph:

[0028] While Fig. 2 shows a valve bolt in which the circumferential surface portion 36,

adjoining the edge 34 inside the hollow throat 18, is oriented at an angle of inclination α of

approximately 60° to the center axis 22 of the valve bolt 6, and the fuel therefore strikes the

inner wall 24 of the outflow bore 26 rather steeply, and thus a relatively large amount of fuel

is directed back in the direction of the valve gap [[28]] 12, Figs. 3 and 4 show two valve bolts

6 in which this angle of inclination α is approximately 35° and approximately 20°,

respectively, and correspondingly less fuel is therefore directed back in the direction of the

valve gap [[28]] 12, forming an eddy 34.

Page 10, please add the following new paragraph after paragraph [0031]:

[0032] The foregoing relates to a preferred exemplary embodiment of the invention, it being

understood that other variants and embodiments thereof are possible within the spirit and

scope of the invention, the latter being defined by the appended claims.

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